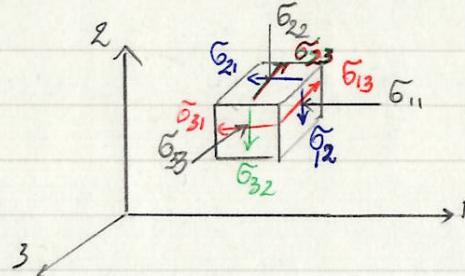
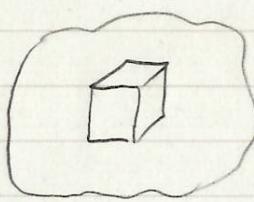


STRESS



- Pick up a point in a continuum body to investigate the stress in this point

- Define the axis

- On each plane, there are 3 stresses: 1 normal & 2 shear

Rule: $\tilde{\sigma}_{ij}$ i : the name of the plane which the stress acts
 j : the direction of the stress

- Put the components of stress in 1 tensor:

$$\tilde{\sigma} = \begin{bmatrix} \tilde{\sigma}_{11} & \tilde{\sigma}_{12} & \tilde{\sigma}_{13} \\ \tilde{\sigma}_{21} & \tilde{\sigma}_{22} & \tilde{\sigma}_{23} \\ \tilde{\sigma}_{31} & \tilde{\sigma}_{32} & \tilde{\sigma}_{33} \end{bmatrix}$$

Meaning of stress tensor:

If we want to check stresses on a specific plane, we only need to use the linear transformation of the normal unit vector of that plane by stress tensor

For ex: plane 1 with $\vec{i} = (1, 0, 0)$

$$\begin{pmatrix} \tilde{\sigma}_{11} & \tilde{\sigma}_{12} & \tilde{\sigma}_{13} \\ \tilde{\sigma}_{21} & \tilde{\sigma}_{22} & \tilde{\sigma}_{23} \\ \tilde{\sigma}_{31} & \tilde{\sigma}_{32} & \tilde{\sigma}_{33} \end{pmatrix} \begin{pmatrix} 1 \\ 0 \\ 0 \end{pmatrix} = \begin{pmatrix} \tilde{\sigma}_{11} \\ \tilde{\sigma}_{12} \\ \tilde{\sigma}_{13} \end{pmatrix}$$

Conservation of momentum:

$$\tilde{\sigma}_{12} = \tilde{\sigma}_{21} ; \quad \tilde{\sigma}_{13} = \tilde{\sigma}_{31} ; \quad \tilde{\sigma}_{23} = \tilde{\sigma}_{32}$$

Mean stress:

The summation of $\tilde{\sigma}_{11}$, $\tilde{\sigma}_{22}$, $\tilde{\sigma}_{33}$ has a strong relation with volumetric behavior:

→ Set:

$$P = \frac{1}{3} (\tilde{\sigma}_{11} + \tilde{\sigma}_{22} + \tilde{\sigma}_{33}) = \frac{1}{3} \tilde{\sigma}_{rr} \quad (\text{Mean stress})$$

3D Stress Tensor

① Principle stresses - eigen values

General stress tensor can be rotated to find a new stress tensor with principle stresses σ_1, σ_2 and no shear stress.

$$\begin{pmatrix} \tilde{\sigma}_{11} & \tilde{\sigma}_{12} \\ \tilde{\sigma}_{21} & \tilde{\sigma}_{22} \end{pmatrix} \rightarrow \begin{pmatrix} \sigma_1 & 0 \\ 0 & \sigma_2 \end{pmatrix}$$

Notice that what we have done is that we found diagonal matrix with eigen value in it (tensor with eigen value). Eigen values are in fact the principle stress

② 3D case:

A symmetric stress tensor: $\begin{pmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{21} & \sigma_{22} & \sigma_{23} \\ \sigma_{31} & \sigma_{32} & \sigma_{33} \end{pmatrix}$

If these are real numbers, this tensor must have 3 real eigen values or 3 principle stresses & 3 corresponding eigen vectors which are orthogonal to each other.

③ Find eigen value:

$$M\tilde{x} = \lambda \tilde{x} \Rightarrow (M - \lambda I)x = 0$$

eigen vector eigen value $\Rightarrow |M - \lambda I| = 0 \Rightarrow \lambda$

For example:

$$\left| \begin{pmatrix} a_{11} & a_{12} & a_{13} \\ a_{21} & a_{22} & a_{23} \\ a_{31} & a_{32} & a_{33} \end{pmatrix} - \lambda I \right| = 0 \Leftrightarrow \begin{vmatrix} a_{11} - \lambda & a_{12} & a_{13} \\ a_{21} & a_{22} - \lambda & a_{23} \\ a_{31} & a_{32} & a_{33} - \lambda \end{vmatrix} = 0$$

$$\Leftrightarrow (a_{11} - \lambda) \begin{vmatrix} a_{22} - \lambda & a_{23} \\ a_{32} & a_{33} - \lambda \end{vmatrix} - a_{12} \begin{vmatrix} a_{21} & a_{23} \\ a_{31} & a_{33} - \lambda \end{vmatrix} + a_{13} \begin{vmatrix} a_{21} & a_{22} - \lambda \\ a_{31} & a_{32} \end{vmatrix} = 0$$

\Rightarrow Find $\lambda_1, \lambda_2, \lambda_3$

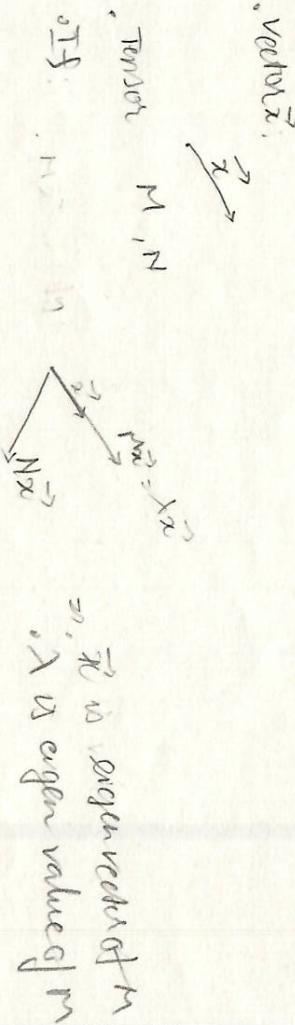
④ Find eigen vector:

$$(\tilde{\sigma} - \lambda I)x = 0 \Rightarrow \tilde{x}_1, \tilde{x}_2, \tilde{x}_3$$

Note: ① $\tilde{x}_1 \perp \tilde{x}_2$; $\tilde{x}_1 \perp \tilde{x}_3$; $\tilde{x}_2 \perp \tilde{x}_3$

② $\lambda_1 + \lambda_2 + \lambda_3 = \text{trace}(M)$

③ $\lambda_1 \cdot \lambda_2 \cdot \lambda_3 = \det(M)$



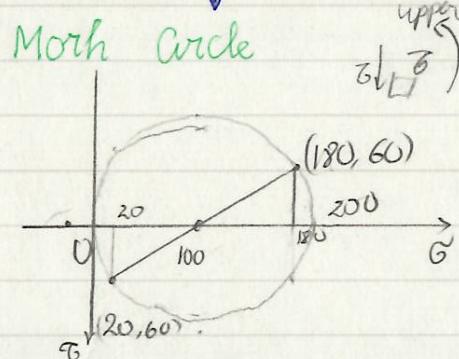
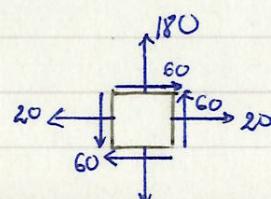
STRESS TENSOR

Example:

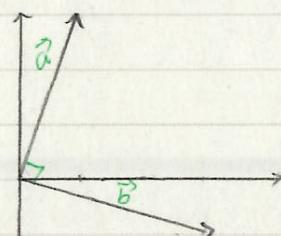
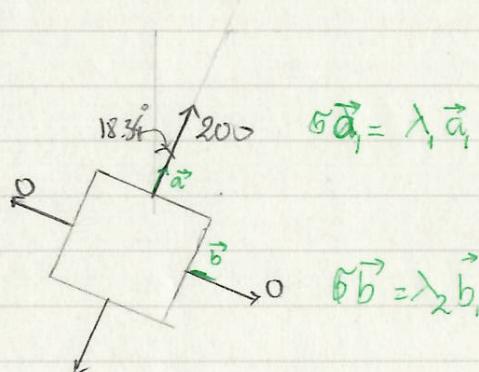
$$\sigma = \begin{pmatrix} 20 & 60 \\ 60 & 80 \end{pmatrix} \text{ MPa}$$

Find

- 1) Principle stresses?
- 2) Maximum shear stress



- Principle stress: 0, 200 MPa
- Maximum shear stress: 100 MPa
- $\tan 2\theta = \frac{60}{80} = \frac{3}{4} \Rightarrow \theta = 18.45^\circ$



Eigen value & Eigen vector

① Eigen value (principle stress)

$$\sigma - \lambda I = 0$$

$$\Rightarrow \begin{pmatrix} 20-\lambda & 60 \\ 60 & 80-\lambda \end{pmatrix} = 0$$

$$\Leftrightarrow (20-\lambda)(80-\lambda) - 60^2 = 0$$

$$\Leftrightarrow \begin{cases} \lambda_1 = 0 \\ \lambda_2 = 200 \end{cases}$$

Note: Characteristic of eigen value of symmetric tensor

$$① \lambda_1 + \lambda_2 = \text{trace } (\sigma)$$

$$② \lambda_1 \cdot \lambda_2 = \det(\sigma)$$

③ Eigen vector: Principle direction:

$$\lambda = 0$$

$$\Rightarrow \begin{pmatrix} 20 & 60 \\ 60 & 80 \end{pmatrix} \begin{pmatrix} x \\ y \end{pmatrix} = 0 \cdot \begin{pmatrix} x \\ y \end{pmatrix}$$

$$\Rightarrow 20x + 60y = 0$$

$$\Rightarrow x + 3y = 0$$

$$\Rightarrow \text{Eigen vector } \vec{a} = \begin{pmatrix} 3/\sqrt{10} \\ -1/\sqrt{10} \end{pmatrix} \text{ (unit vector)}$$

$$\lambda = 200$$

$$\Rightarrow \begin{pmatrix} 20 & 60 \\ 60 & 80 \end{pmatrix} \begin{pmatrix} x \\ y \end{pmatrix} = 200 \begin{pmatrix} x \\ y \end{pmatrix}$$

$$\Rightarrow 60x = 180y$$

$$\Rightarrow \text{Eigen vector } \vec{b} = \begin{pmatrix} 3/\sqrt{10} \\ 1/\sqrt{10} \end{pmatrix} \text{ (unit vector)}$$

 Note: $\vec{a} \perp \vec{b}$ ($a \cdot b = 0$)

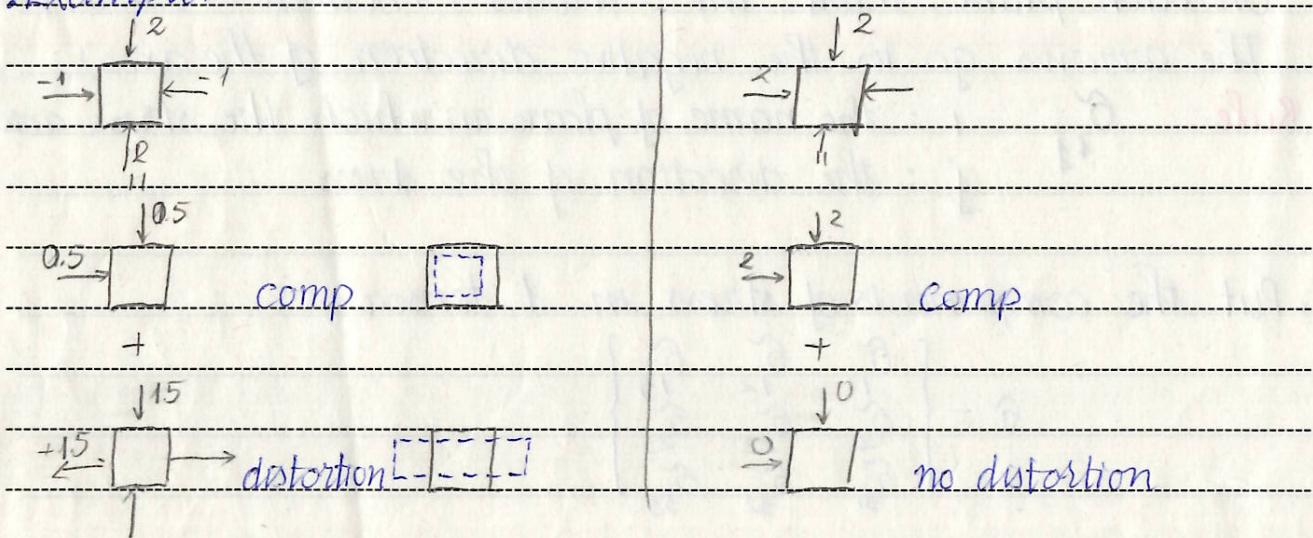


Constitutive Modelling

Deviator Stress:

$$\begin{array}{l} \text{Total stress} \quad \text{Mean stress} \quad \text{Distortional stress tensor} \\ \left(\begin{matrix} \tilde{\sigma}_{11} & \tilde{\sigma}_{12} & \tilde{\sigma}_{13} \\ \tilde{\sigma}_{21} & \tilde{\sigma}_{22} & \tilde{\sigma}_{23} \\ \tilde{\sigma}_{31} & \tilde{\sigma}_{32} & \tilde{\sigma}_{33} \end{matrix} \right) - \left(\begin{matrix} p & 0 & 0 \\ 0 & p & 0 \\ 0 & 0 & p \end{matrix} \right) = \left(\begin{matrix} \tilde{\sigma}_{11}-p & \tilde{\sigma}_{12} & \tilde{\sigma}_{13} \\ \tilde{\sigma}_{21} & \tilde{\sigma}_{22}-p & \tilde{\sigma}_{23} \\ \tilde{\sigma}_{31} & \tilde{\sigma}_{32} & \tilde{\sigma}_{33}-p \end{matrix} \right) = S_{ij} = \text{Deviator stress tensor} \\ \Leftrightarrow \tilde{\sigma}_{ij} - p = S_{ij} \end{array}$$

Example:



Magnitude of deviator stress tensor:

- Magnitude of vector: $\|a\| = \sqrt{a_1^2 + a_2^2 + a_3^2} = \sqrt{a_i a_i}$
- Magnitude of $S_{ij} : \sqrt{S_{ij} S_{ij}}$

Deviator stress

$$\text{Definition: } q = \sqrt{\frac{3}{2} \sqrt{s_{11}s_{33}}} = \sqrt{\frac{3}{2}} \sqrt{s_{11} + s_{12} + \dots + s_{33}}$$

Triaxial test: $\sigma_{ij} = 0$ ($i \neq j$) ; $\sigma_{22} = \sigma_{33} = \sigma_3$

$$q = \frac{\sqrt{3}}{2} \sqrt{\left(2\tilde{\sigma}_{11} - 2\tilde{\sigma}_{33}\right)^2 + 2\left(\frac{\tilde{\sigma}_{33} - \tilde{\sigma}_{11}}{3}\right)^2} = \left|\tilde{\sigma}_{11} - \tilde{\sigma}_{33}\right| = \tilde{\sigma}_{11} - \tilde{\sigma}_{33}$$

$$= \frac{\sqrt{3}}{2} \cdot \sqrt{\frac{2}{3}} \left|\tilde{\sigma}_1 - \tilde{\sigma}_3\right|$$

STRESS TENSOR

① Determine $\frac{\partial p}{\partial \tilde{G}_{ij}}$:

$$\frac{\partial p}{\partial \tilde{G}_{ij}} = \frac{\partial (\frac{1}{3} \tilde{G}_{kk})}{\partial \tilde{G}_{ij}} = \frac{1}{3} \frac{\partial \tilde{G}_{kk}}{\partial \tilde{G}_{ij}} = \frac{1}{3} \delta_{ik} \delta_{jk} = \frac{1}{3} \delta_{ik} \delta_{kj} = \frac{1}{3} \delta_{ij}$$

② Determine $\frac{\partial q}{\partial \tilde{G}_{ij}}$:

$$S_{ij} = \tilde{G}_{ij} - p S_{ij} \Rightarrow q = \sqrt{\frac{3}{2} S_{ij} S_{ij}} \Rightarrow \frac{\partial q}{\partial \tilde{G}_{ij}} = \frac{\partial \sqrt{\frac{3}{2} S_{ij} S_{ij}}}{\partial \tilde{G}_{ij}}$$

$$\begin{aligned} \frac{\partial q}{\partial S_{ij}} &= \frac{\partial \sqrt{\frac{3}{2} S_{kl} S_{kl}}}{\partial S_{ij}} = \sqrt{\frac{3}{2}} \frac{\partial (\tilde{S}_{kl} \tilde{S}_{kl})^{\frac{1}{2}}}{\partial S_{ij}} \\ &= \sqrt{\frac{3}{2}} \frac{1}{2} \frac{1}{\sqrt{S_{kl} S_{kl}}} \frac{\partial (S_{kl} S_{kl})}{\partial S_{ij}} \\ &= \sqrt{\frac{3}{2}} \frac{1}{2} \frac{1}{\|S_{kl}\|} 2 S_{kl} = \sqrt{\frac{3}{2}} \frac{S_{kl}}{\|S_{kl}\|} \end{aligned}$$

$$\begin{aligned} \frac{\partial S_{ij}}{\partial \tilde{G}_{kl}} &= \frac{\partial (\tilde{G}_{ij} - p S_{ij})}{\partial \tilde{G}_{kl}} = \frac{\partial \tilde{G}_{ij}}{\partial \tilde{G}_{kl}} - \frac{\partial p S_{ij}}{\partial \tilde{G}_{kl}} \\ &= \delta_{ki} \delta_{lj} - \frac{\partial (\frac{1}{3} \tilde{G}_{mm} \delta_{ij})}{\partial \tilde{G}_{kl}} \quad \delta_{km} \delta_{pl} = \delta_{kl} \\ &= \delta_{ki} \delta_{lj} - \frac{1}{3} \delta_{ij} \frac{\partial \tilde{G}_{mm}}{\partial \tilde{G}_{kl}} = \delta_{ki} \delta_{lj} - \frac{1}{3} \delta_{ij} \delta_{kl} \end{aligned}$$

$$\begin{aligned} \frac{\partial q}{\partial \tilde{G}_{kl}} &= \frac{\partial q}{\partial S_{ij}} \frac{\partial S_{ij}}{\partial \tilde{G}_{kl}} = \sqrt{\frac{3}{2}} \frac{S_{kl}}{\|S_{kl}\|} \cdot \left(\delta_{ki} \delta_{lj} - \frac{1}{3} \delta_{ij} \delta_{kl} \right) \\ &= \sqrt{\frac{3}{2}} \frac{1}{\|S_{kl}\|} \left(\underbrace{\delta_{ik} \delta_{kl} \delta_{lj}}_{S_{ij}} - \frac{1}{3} \underbrace{\delta_{kl} \delta_{lk} \delta_{ij}}_{S_{kk} \rightarrow 0} \right) \end{aligned}$$